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MODAL RESPONSE CONTRIBUTIONS

Modal Expansion of Excitation Vector $\mathbf{p}(t) = \mathbf{s}p(t)$

A common loading case is the one in which the applied forces $p_j(t)$ have the same time variation p(t), and their <u>spatial distribution</u> is defined by **s**, <u>independent of time</u>.

$$\mathbf{p}(t) = \mathbf{s}p(t)$$

We **expand** the vector **s** in its modal components as:

$$\mathbf{s} = \sum_{r=1}^{N} \mathbf{s}_r = \sum_{r=1}^{N} \Gamma_r \mathbf{m} \mathbf{\phi}_r$$

Pre-multiplying by $\boldsymbol{\phi}_n^T$ and utilizing the orthogonality property of the modes, we obtain:

$$\Gamma_n = rac{\mathbf{\phi}_n^T \mathbf{s}}{M_n}$$

Modal

Participation Factor

It is evident that Γ_n is <u>not</u> independent of how the mode is normalized.

The contribution of the n^{th} mode to **s** is:

$$\mathbf{s}_n = \Gamma_n \mathbf{m} \mathbf{\phi}_n$$

which is independent of how the modes are normalized.

The inertia forces, associated with the n^{th} mode, are:

$$(\mathbf{f}_I)_n = -\mathbf{m}\ddot{\mathbf{u}}_n(t) = -\mathbf{m}\mathbf{\phi}_n\ddot{q}_n(t)$$

Clearly, the spatial distribution of the inertia forces $(\mathbf{f}_I)_n$ associated with the n^{th} mode, is the same as that of \mathbf{s}_n .

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The expansion,

$$\mathbf{s} = \sum_{r=1}^{N} \mathbf{s}_r = \sum_{r=1}^{N} \Gamma_r \mathbf{m} \mathbf{\phi}_r$$

has two useful properties:

(1) The force vector $\mathbf{s}_n p(t)$ produces response only in the n^{th} mode but no response in any other mode.

Proof:

Generalized force associated with $\mathbf{s}_n p(t)$:

$$P_r(t) = \mathbf{\phi}_r^T \mathbf{s}_n p(t) = \Gamma_r \underbrace{(\mathbf{\phi}_r^T \mathbf{m} \mathbf{\phi}_n)}_{M_n \delta_{nr}} p(t) = \Gamma_r M_n \delta_{nr} p(t)$$
$$[\delta_{nr}: Kronecker\ delta]$$

(2) The dynamic response of the nth mode is entirely due to the partial force vector $\mathbf{s}_n p(t)$.

Proof:

Generalized force associated with $\mathbf{sp}(t)$:

$$P_n(t) = \mathbf{\phi}_n^T \mathbf{s} p(t) = \sum_{r=1}^N \Gamma_r(\mathbf{\phi}_n^T \mathbf{m} \mathbf{\phi}_r) p(t) = \Gamma_n M_n p(t)$$

Evidently, the n^{th} mode generalized force associated with the complete force vector $\mathbf{s}p(t)$ is the same as the n^{th} mode generalized force associated with the partial force vectors $\mathbf{s}_n p(t)$.

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Modal Analysis for
$$p(t) = sp(t)$$

We have derived *N* uncoupled equations:

$$\ddot{q}_n(t) + 2\xi_n \omega_n \dot{q}_n(t) + \omega_n^2 q_n(t) = \frac{P_n(t)}{M_n} \quad (n = 1, 2, \dots, N)$$
where:
$$P_n(t) = \mathbf{\Phi}_n^T \mathbf{p}(t) = \mathbf{\Phi}_n^T \mathbf{s} p(t) = \Gamma_n M_n p(t)$$

Therefore:

$$\ddot{q}_n(t) + 2\xi_n \omega_n \dot{q}_n(t) + \omega_n^2 q_n(t) = \Gamma_n p(t)$$

We can write the solution $q_n(t)$ as follows:

$$\begin{vmatrix} \ddot{D}_n(t) + 2\xi_n \omega_n \dot{D}_n(t) + \omega_n^2 D_n(t) = p(t) \\ \text{and} \quad q_n(t) = \Gamma_n D_n(t) \end{vmatrix}$$

Contribution of the n^{th} mode to $\mathbf{u}(t)$:

$$\mathbf{u}_n(t) = \mathbf{\phi}_n q_n(t) = \Gamma_n \mathbf{\phi}_n D_n(t)$$

Equivalent static forces $\mathbf{f}_{Sn}(t)$:

$$\mathbf{f}_{Sn}(t) = \mathbf{k}\mathbf{u}_n(t) = \Gamma_n \underbrace{\mathbf{k}\mathbf{\phi}_n}_{\omega_n^2 \mathbf{m}\mathbf{\phi}_n} D_n(t) = \underbrace{\Gamma_n \mathbf{m}\mathbf{\phi}_n}_{\mathbf{s}_n} [\omega_n^2 D_n(t)]$$

$$\Rightarrow \underbrace{\mathbf{f}_{Sn}(t) = \mathbf{s}_n[\omega_n^2 D_n(t)]}$$

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Equivalent Static Forces $\mathbf{f}_{Sn}(t)$:

$$\mathbf{f}_{Sn}(t) = \mathbf{s}_n[\omega_n^2 D_n(t)]$$

The n^{th} mode contribution $r_n(t)$ to any response quantity r(t) is determined by static analysis of the structure subjected to forces $\mathbf{f}_{Sn}(t)$.

Let:

$$r_n^{st}$$
 = the static value of response quantity r due to external forces s_n

Then:

$$r(t) = \sum_{n=1}^{N} r_n(t) = \sum_{n=1}^{N} r_n^{st} [\omega_n^2 D_n(t)]$$

The modal analysis procedure presented above provides a basis for identifying and understanding the factors that influence the relative modal contributions to the response.

NOTE:

Although we loosely refer to s_n as forces, they are dimensionless because p(t) has units of force.

Thus, r_n^{st} does <u>not</u> have the same units as r, but equation $r_n(t) = r_n^{st} [\omega_n^2 D_n(t)]$ gives the correct units for r_n and equation $r(t) = \sum_{n=1}^N r_n(t)$ for r.

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Modal Contribution Factors

Let:

$$r^{St}$$
 = the static value of response quantity r due to external forces s

Contribution of the n^{th} mode to response quantity r:

$$r_n(t) = r_n^{st} [\omega_n^2 D_n(t)] = r^{st} \left(\frac{r_n^{st}}{r^{st}}\right) [\omega_n^2 D_n(t)]$$

$$= r^{st} \bar{r}_n [\omega_n^2 D_n(t)]$$
where: $\bar{r}_n = \left(\frac{r_n^{st}}{r^{st}}\right)$ modal contribution factor

Properties of modal contribution factors \bar{r}_n :

- (1) They are dimensionless;
- (2) They are independent of how modes are normalized;
- (3) The sum of the modal contribution factors over all modes is unity, that is:

$$\sum_{n=1}^{N} \bar{r}_n = 1$$

Property #3 follows by the following reasoning:

$$\mathbf{s} = \sum_{n=1}^{N} \mathbf{s}_{n} \quad \Rightarrow \quad r^{st} = \sum_{n=1}^{N} r_{n}^{st} \quad \Rightarrow \quad 1 = \sum_{n=1}^{N} \underbrace{\left(\frac{r_{n}^{st}}{r_{n}^{st}}\right)}_{\overline{r_{n}}}$$

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MODAL ANALYSIS: SUMMARY

Dynamic response of an MODF system with classical damping to external forces $\mathbf{p}(t)$:

- (1) Define the **structural properties**:
 - (a) Determine the mass matrix \mathbf{m} and stiffness matrix \mathbf{k} ;
 - (b) Estimate the modal damping ratios ξ_n .
- (2) Determine the natural frequencies ω_n and modes ϕ_n ; [i.e., solve the eigenvalue problem: $(\mathbf{k} \omega^2 \mathbf{m}) \phi = \mathbf{0}$]
- (3) Compute the response in each mode by following the steps:
 - (a) Set up the following *N* uncoupled equations $(n = 1, 2, \dots, N)$:

and solve for $q_n(t)$.

- (b) Compute the **nodal displacements** $\mathbf{u}_n(t) = \mathbf{\phi}_n q_n(t)$
- (c) Compute the **element forces** associated with $u_n(t)$ by implementing the **equivalent static** force method:

$$\mathbf{f}_{Sn}(t) = \mathbf{k}\mathbf{u}_n(t) \implies \mathbf{f}_{Sn}(t) = \omega_n^2 \mathbf{m} \mathbf{\phi}_n q_n(t) \quad (n = 1, 2, \dots, N)$$

(4) Combine the contributions of all the modes to determine the response:

$$\left. \begin{array}{l} nodal \\ displacements \end{array} \right\} \quad \mathbf{u}(t) = \sum_{n=1}^{N} \mathbf{u}_n(t) = \sum_{n=1}^{N} \boldsymbol{\phi}_n q_n(t) \\ e. \, g. \quad \\ element \\ forces \end{array} \right\}$$

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MODAL ANALYSIS FOR
$$\mathbf{p}(t) = \mathbf{s}p(t)$$

After we determine:

- the structural properties (i.e., $\mathbf{m} \& \mathbf{k}, \xi_n$).
- the **vibration properties** (i.e., $\omega_n \& \mathbf{\phi}_n$).

we expand the force distribution s into its modal components s_n , *i.e.*,

$$\mathbf{s} = \sum_{n=1}^{N} \mathbf{s}_n = \sum_{n=1}^{N} \{\Gamma_n \mathbf{m} \mathbf{\phi}_n\}$$

The contribution $r_n(t)$ of the n^{th} mode to the dynamic response is obtained by multiplying the results of two analyses:

- (1) **Static analysis** of the structure subjected to external forces \mathbf{s}_n $(n = 1, 2, \dots, N)$.
- (2) **Dynamic analysis** of the n^{th} mode SDOF system excited by the force p(t):

$$\ddot{D}_n(t) + 2\xi_n \omega_n \dot{D}_n(t) + \omega_n^2 D_n(t) = p(t) \quad (n = 1, 2, \dots, N)$$

$$q_n(t) = \Gamma_n D_n(t)$$

Combining the modal responses gives the dynamic response of the structure.

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Conceptual explanation of modal analysis

Mode	Static Analysis of	Dynamic Analysis of	Modal Contribution to		
	Structure	SDF System	Dynamic Response		
n	Forces s _n	Unit mass $p(t) \longrightarrow D_n(t)$ ω_n, ζ_n	$r_n(t) = r_n^{\rm st} \left[\omega_n^2 \ D_n(t) \right]$		

Modal Responses: n = 1, 2, ...N

$$r_n(t) = r_n^{\text{st}} \left[\omega_n^2 D_n(t)\right]$$

Total Response

$$r(t) = \sum_{n=1}^{N} r_n(t) = \sum_{n=1}^{N} r_n^{\text{st}} [\omega_n^2 D_n(t)]$$

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Modal Responses and Required Number of Modes

Let:

$$D_{no} \stackrel{\text{\tiny def}}{=} \max_{t} |D_n(t)|$$

Then, the corresponding value of $r_n(t)$ is:

$$r_{no} = \underbrace{r^{st}\bar{r}_n}_{r_n^{st}} \omega_n^2 D_{no}$$

Let:

$$R_{dn} = \frac{D_{no}}{\left(D_{n,st}\right)_o}$$
 Dynamic Response Factor

where:

$$(D_{n,st})_o = \max_t \underbrace{|D_{n,st}(t)|}_{static \ response}$$

$$= \left(\frac{\max_t |p(t)|}{\omega_n^2}\right)$$

$$= \left(\frac{p_o}{\omega_n^2}\right)$$

The <u>static response</u> $D_{n,st}(t)$ is obtained from $\ddot{D}_n(t) + 2\xi_n\omega_n\dot{D}_n(t) + \omega_n^2D_n(t) = p(t)$, by dropping the $\ddot{D}_n \& \dot{D}_n$ terms.

Therefore:

$$r_{no} = r^{st} \bar{r}_n p_o R_{dn}$$

The algebraic sign of r_{no} is the same as that of $r_n^{st} \stackrel{\text{def}}{=} r^{st} \bar{r}_n$ because R_{dn} is positive by definition.

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Peak Modal Response

$$r_{no} = r^{st} \bar{r}_n p_o R_{dn}$$

Peak modal response is the product of four quantities:

- (1) R_{dn} (=dimensionless) <u>dynamic response factor</u> for the n^{th} mode SDOF excited by force p(t)
- (2) p_o , the **peak value** of p(t)
- (3) \bar{r}_n (=dimensionless) <u>modal contribution factor</u> for response quantity r
- (4) r^{st} , the <u>static value</u> of r due to external forces s

Clearly:

 r^{st} & \bar{r}_n : depend on the spatial distribution s of the applied forces but are independent of the time variation p(t) of the applied forces.

 R_{dn} : depends on p(t), but is independent of s.

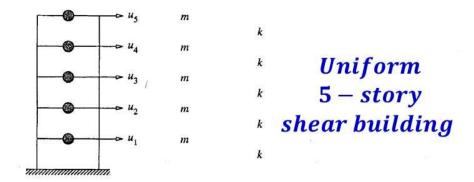
 $\bar{r}_n \& R_{dn}$ influence the relative response contributions of the various vibration modes, and hence the minimum number of modes that should be included in the dynamic analysis.

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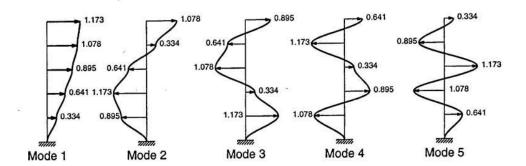
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EXAMPLE:



The mass and stiffness matrices of the structure are

Natural vibration modes of five-story shear building



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The <u>relative contributions</u> of the various modes to the response,

and

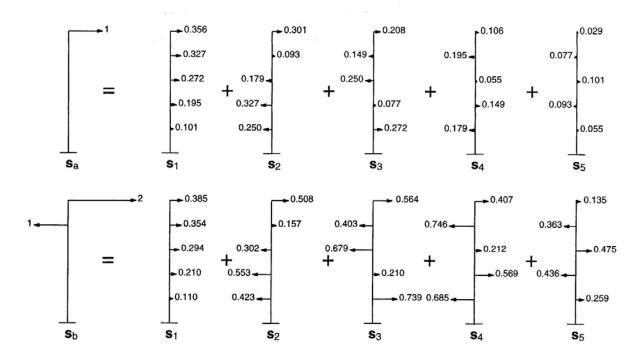
the <u>number of modes</u> that **should be included** in dynamic analysis depend on the following **two influencing factors**:

- (1) The *modal contribution factor* \bar{r}_n , which depends on the **spatial** distribution of forces.
- (2) The *dynamic response factor* R_{dn} , which is controlled by the **time variation** of forces.

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Modal expansion of excitation vectors \mathbf{s}_a and \mathbf{s}_b



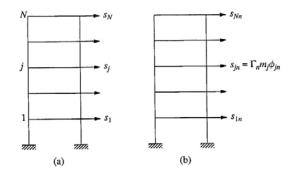
Observation:

The contributions of the <u>higher</u> modes to s are larger for s_b than for s_a , suggesting that these modes (*i.e.*, the higher modes) may contribute more to the response if the force distribution is s_b than if it is s_a .

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Determination of the *modal contribution factors* for:

- The base shear V_b (i)
- The roof (Nth floor) displacement u_N (ii)

$$V_{bn}^{st} = \sum_{j=1}^{N} s_{jn} = \Gamma_n \sum_{j=1}^{N} m_j \phi_{jn} =$$

$$\mathbf{u}_n^{st} = \mathbf{k}^{-1} \mathbf{s}_n = \mathbf{k}^{-1} \Gamma_n \mathbf{m} \boldsymbol{\phi}_n = \frac{\Gamma_n}{\omega_n^2} \boldsymbol{\phi}_n$$

(Recall that: $\mathbf{k}\mathbf{\phi}_n = \omega_n^2 \mathbf{m}\mathbf{\phi}_n \quad \Longleftrightarrow \quad \frac{1}{\omega_n^2}\mathbf{\phi}_n = \mathbf{k}^{-1} \mathbf{m}\mathbf{\phi}_n$)

Therefore:

$$u_{Nn}^{st} = \frac{\Gamma_n}{\omega_n^2} \phi_{Nn}$$
 $\begin{array}{c} roof~(N^{th}~floor) \\ displacement \\ for~n^{th}~mode \end{array}$

 $V_h^{st} \& u_N^{st}$ are obtained from a static analysis of the building subjected to (lateral) forces S.

We define:

$$\bar{V}_{bn} = \frac{V_{bn}^{st}}{V_b^{st}} \quad \& \quad \bar{u}_{Nn} = \frac{u_{Nn}^{st}}{u_N^{st}}$$

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TABLE 12.11.1 MODAL AND CUMULATIVE CONTRIBUTION FACTORS

	Force Distribution, s_a			Force Distribution, s_b				
	Roof Displacement		Base Shear		Roof Displacement		Base Shear	
Mode n or Number of Modes, J	\overline{u}_{5n}	$\sum_{n=1}^{J} \overline{u}_{5n}$	\overline{V}_{bn}	$\sum_{n=1}^{J} \overline{V}_{bn}$	\overline{u}_{5n}	$\sum_{n=1}^{J} \overline{u}_{5n}$	\overline{V}_{bn}	$\sum_{n=1}^{J} \overline{V}_{bn}$
1	0.880	0.880	1.252	1.252	0.792	0.792	1.353	1.353
2	0.087	0.967	-0.362	0.890	0.123	0.915	-0.612	0.741
3	0.024	0.991	0.159	1.048	0.055	0.970	0.431	1.172
4	0.008	0.998	-0.063	0.985	0.024	0.994	-0.242	0.930
5	0.002	1.000	0.015	1.000	0.006	1.000	0.070	1.000

Observations:

(1) The *modal contribution factors* are <u>larger</u> for base shear than for roof displacement.

Higher modes contribute more to base shear (and other element forces) **than to roof displacement** (and other floor displacements)

(2) The *modal contribution factors* for higher modes are <u>larger</u> for force distribution \mathbf{s}_b than for \mathbf{s}_a .

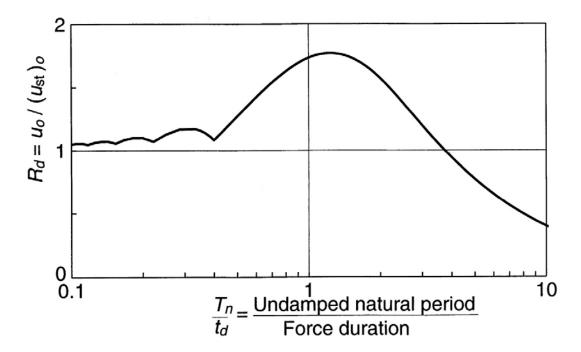
Higher modes contribute more to a response in the \mathbf{S}_b case.

The modal expansion of $S_a & S_b$ suggests the same conclusion.

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Dynamic response factor for half-cycle sine pulse force



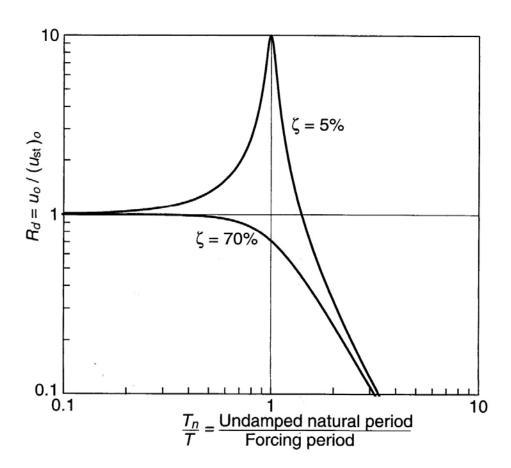
How R_{dn} (= the value of R_d for the n^{th} mode) for a given excitation p(t) varies with n

depends on

where the natural periods T_n fall on the period scale.

In the case of **pulse excitation**, the above FIGURE shows that R_{dn} varies over a narrow range for a wide range of T_n and could have similar values for several modes.

Dynamic response factor for harmonic force



For **lightly damped systems** subjected to harmonic excitation (Ω) , R_{dn} is especially large for modes with natural period T_n close to the forcing period T, and perhaps these are the only modes that need be included in modal analysis unless the modal contribution factors \bar{r}_n for these modes are much smaller than for some other modes.

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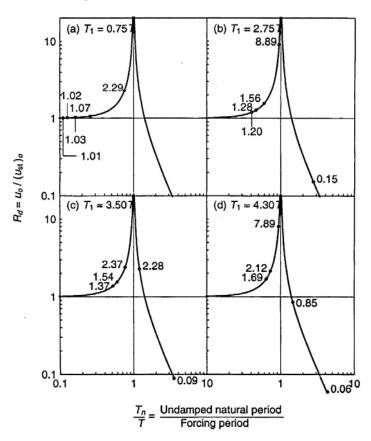
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TABLE 12.11.1 MODAL AND CUMULATIVE CONTRIBUTION FACTORS

	Force Distribution, s_a			Force Distribution, s_b				
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Mode n or Number of Modes, J	\overline{u}_{5n}	$\sum_{n=1}^{J} \overline{u}_{5n}$	\overline{V}_{bn}	$\sum_{n=1}^{J} \overline{V}_{bn}$	\overline{u}_{5n}	$\sum_{n=1}^{J} \overline{u}_{5n}$	\overline{V}_{bn}	$\sum_{n=1}^{J} \overline{V}_{bn}$
1	0.880	0.880	1.252	1.252	0.792	0.792	1.353	1.353
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3	0.024	0.991	0.159	1.048	0.055	0.970	0.431	1.172
4	0.008	0.998	-0.063	0.985	0.024	0.994	-0.242	0.930
5	0.002	1.000	0.015	1.000	0.006	1.000	0.070	1.000

Dynamic response factors R_{dn}



In judging the contribution of a natural mode to the dynamic response of a structure, it is necessary to consider the combined effects of the modal contribution factor \bar{r}_n and the dynamic response factor R_{dn} .